NUMERICAL STUDY OF SADDLE-SHAPED VOID FRACTION PROFILES EFFECT ON THERMAL HYDRAULIC PARAMETERS OF THE CHANNEL WITH TWO-PHASE FLOW USING OPENFOAM AND COMPARISON WITH EXPERIMENTS

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Introduction

- Two- and *multiphase flows* with bubble structure is widely presented in traditional and *nuclear power industry*, in chemical and technological production, pipeline transport of hydrocarbons, metallurgical and other equipment. Regimes of normal operation for such systems are justified by experimental and numerical methods based on regulatory and reference data.

- The models presented in the majority of the *one-dimensional codes* not always adequately enough describe gas distribution via radius of the channel and don’t take into account abnormal growth of friction at the wall of the channel, and for this reason they are in the process of active verification.

- Implementation of CFD codes for multiphase flow simulation with “Euler-Euler” approach allows to overcome shortcoming of the one-dimensional semi-empirical codes.

Purpose of the work

- The present work is dedicated to verification of numerical model in standard solver of open-source CFD code OpenFOAM for two-phase flow simulation – twoPhaseEulerFoam, and to determination of the “baseline” model parameters. The investigation of heterogeneous distributions of non-equilibrium coolant flow, which leads to abnormal friction increase of the channel in two-phase adiabatic “water-gas” flows with low void fractions, presented.

- There are a significant number of works in open literature, dedicated to two-phase flow modeling with CFD codes have the following distinctive features:
  - authors use modified solvers with two-phase models;
  - for verification they use international experimental data for void fraction distribution;
  - authors basically don’t consider effects of abnormal friction increase.

- On contrary, during this work authors used default OpenFOAM solver for velocities and void-fraction profiles calculation; the results of simulation were compared against Russian experimental data including comparison of friction at the wall. As a result the friction coefficient of channels walls were obtained in the wide range of gas flow ratio $\beta$. 

Development of *twoPhaseEulerFoam*

- **OpenFOAM 1.7.x (2010) twoPhaseEulerFoam**
  - Euler-Euler solver for turbulent two-phase flows
  - Constant material properties
  - Constant dispersed phase diameter
  - No bubble specific drag models, turbulent dispersion or wall lubrication force
  - No heat or mass transfer
  - Hard coded k-eps turbulence model

- **2.1.0 (2011)**
  - Temperature base heat transfer solution, compressibility, non-uniform diameter

- **2.1.1 (2012)**
  - Improved void fraction solution algorithm (MULES)

- **2.2.0 (2013)**
  - Support for multiphase thermodynamics, enthalpy based energy solution

- **2.3.0 & 2.3.1 (2014)**
  - Consolidation, new runtime selectable interfacial and turbulence models. Large selection of closure models

- **3.0 & 3.0.1 (2015)**
  - Boundary condition with heat flux

- **4.0 (2016)**
  - Simplification of sub-models use
Information on experiment

Siberian Branch of Russian Academy of Sciences

Schematic diagram of experimental set-up: 1—pump, 2, 14— orifice meters, 3, 15—differential manometers, 4—mixer, 5—upward channel, 6—regions of visualization, 7—test section, 8—downward channel, 9—adjusting valve, 10—separator, 11—compressed air vessels, 12—filter, 13—heater, 16—heat exchanger, 17—storage tank.

Experimental data
Case set-up

- **inlet:**
  - Uniform velocity profile for liquid and gas phases ($U_g, U_l$)
  - Uniform void fraction profile ($\alpha$)

- **wall:**
  - No-slip BC for liquid phase
  - Slip BC for gas phase
  - Standard wall-function for turbulence parameters

- **outlet:**
  - `zeroGradient` for velocity and turbulence parameters,
  - fixed pressure value
**Interfacial interaction models**

\[ \vec{M}_{ki} = -\vec{M}_{ik} = \vec{M}_D + \vec{M}_L + \vec{M}_{TD} + \vec{M}_W \]

- drag, lift, turbulent dispersion and wall forces:

- Interfacial interaction models in OpenFOAM 2.3.0 (2012):

<table>
<thead>
<tr>
<th>Drag, ( C_d )</th>
<th>Virtual Mass, ( C_{vm} )</th>
<th>Turbulent dispersion:</th>
</tr>
</thead>
<tbody>
<tr>
<td>Tomiyama (2002)</td>
<td>Constant ( C_{vm} )</td>
<td>Bertodano (RPI) (1992)</td>
</tr>
<tr>
<td>Schiller-Nauman (1935)</td>
<td></td>
<td>None</td>
</tr>
<tr>
<td>Constant ( C_d )</td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

- Swarm correction for Drag:
  - Tomiyama (1995)
  - None

- Lift, \( C_l \):
  - Tomiyama (2002)
  - Constant \( C_l \)

- Bubble aspect ratio:
  - Hosokawa & Tomiyama (2009)
  - Vakrushev & Effremov (1970)
  - Wellek (1966)
  - Constant

- Wall lubrication force:
  - Frank (2004)
  - Antal (1991)
  - None
twoPhaseEulerFoam Properties Reader

- Utility to simplify phaseProperties file input
Determining optimal mesh size

- $\alpha = 0$

- $\alpha = 0.05$

- Data [Nakoryakov], mesh 1 – 1000 cells, mesh 2 – 4000 cells, mesh 3 – 10 000 cells

Comparison of simulation results and experimental data for single-phase simulation

Simulation results (1)

Void fraction distribution via radius and height of the channel

\( \frac{r}{r_0} \)

- \( h = 0.1 \text{ m} \)
- \( h = 0.3 \text{ m} \)
- \( h = 0.6 \text{ m} \)
- \( h = 0.05 \text{ m} \)
- \( h = 0.2 \text{ m} \)
- \( h = 0.4 \text{ m} \)
Simulation results (2)

- OpenFOAM 2.3

- «Baseline» model
- $d_b = \text{const} (1 \text{ mm})$
- $T9: \alpha = 0.05; \beta = 0.07$
  Re = 44 000
- $T6: \alpha = 0.07; \beta = 0.15$
  Re = 22 000
- $T19: \alpha = 0.03; \beta = 0.04$
  Re = 110 000
- $T12: \alpha = 0.15; \beta = 0.2$
  Re = 44000

Void fraction distribution via radius
Simulation results (3)

- OpenFOAM 3.0 | «baseline model» + $d_b$ – IATE (0,5 – 1,5 мм) + turbulentBreakUp + kOmegaSST-Sato + randomCoalescence

![Graphs showing simulation and experiment results for different regimes]
Simulation results (4)

Зависимость от размера пузырька, d [мм]
Simulation results (5)

Turbulence suppression in two-phase flow

- k, omega

- Single phase
- Two-phase

Graph showing the comparison between single-phase and two-phase flow dynamics.
Simulation results (6)

- Dependence of shear stress at the wall $\tau$ on gas flow ratio $\beta$

Conclusion

- In the work the result of numerical simulation of void fraction distribution via radius of the channel with two-phase flow with low ($\beta = 0.05 \ldots 0.2$) gas flow ratios are presented.

- In the area of the low void-fractions simulation results demonstrate abnormal increase (six times increase in the maximum) of shear stress in the round channel, which was observed experimentally.

- Based on simulation results the decision was made that OpenFOAM code is adequately simulates saddle-shaped void-fraction profile, which is believed to be the reason of abnormal increase of shear stress in channel at $0 < \beta < 0.2$. 
Thank you for attention!
\[ \mu = \mu^\text{mol} + \mu^\text{turb} + \mu^\text{bub} \]

\[ \mu^\text{bub} = C_B \rho_L \alpha_G d_B |u_G - u_L| \]
Рис. 4. Профили локального газосодержания (а) и скорости жидкости (б) при β = 10%:
1 — Re=990; 2 — Re=2280; 1 — d=1 мм; 2 — 0,5; r, R, m; u, u1, м/с; φ, %
Figure 5.20. experimental and predicted flow variables profiles for bubbly-to-slug transition case F03G03
Fig. 6. Comparison of baseline model simulation results for the profiles of the turbulent kinetic energy with experimental data for different experimental runs from (a) Mohd Akbar et al. (2012), time averaged result from a transient simulation, (b) Liu (1998), steady state simulation (all figures are taken from Rzehak et al. 2014) were also the details.


Fig. 1. Snapshots of simulated instantaneous vapor volume fraction (a & b) and liquid temperature (c & d) for DEBORA5 and DEBORA7, respectively ($r = 0$ m to $r = 0.0096$ m and $z = 3.0$ m to $z = 3.5$ m)
Fig. 2. Radial profiles of predicted velocity, turbulence kinetic energy and velocity fluctuations from different turbulence models compared against single-phase data from experiments H1 and LB1. GL: Gibson and Launder (1978); NR: Naot and Rodi (1982); SSG: Speziale et al. (1991).